2002 GUIDELINE for

SPECIFYING THE THERMAL PERFORMANCE OF COOL STORAGE EQUIPMENT



Guideline T

IMPORTANT

SAFETY RECOMMENDATIONS

It is strongly recommended that the product be designed, constructed, assembled and installed in accordance with nationally recognized safety requirements appropriate for products covered by this guideline.

ARI, as a manufacturers' trade association, uses its best efforts to develop guidelines, employing state-of-the-art and accepted industry practices. However, ARI does not certify or guarantee safety of any products, components or systems designed, tested, rated, installed or operated in accordance with these guidelines or that any tests conducted under its guidelines will be non-hazardous or free from risk.

This guideline supersedes ARI Guideline T-1994.

INTRODUCTION TO THERMAL STORAGE GUIDELINE

Thermal Energy Storage (TES) is a proven technology which enables the use of lower cost, off-peak electricity (usually at night) to produce and store cool energy. This cool energy in storage is used the next day for air-conditioning or process cooling. With TES, relatively small equipment operates at night, reducing the use of expensive electricity during the day.

The types of equipment used in connection with this technology may vary widely.

Unlike most air-conditioning and refrigeration equipment, Thermal Storage Devices have no sustained, steady-state operating point which can be used to characterize the product performance.

Similarly, the usable capacity of a particular Thermal Storage Device may vary appreciably with the application. For example, very high loads discharged over a short period, and/or relatively low discharge temperatures may reduce the usable capacity to a fraction of the nominal value.

These intrinsic characteristics of Thermal Storage Equipment can add complexity to the tasks of rating, selecting and specifying such devices. This guideline has been prepared by the Air-Conditioning and Refrigeration Institute to establish a common, consistent nomenclature and terminology for the industry, and to set forth the minimum performance information that designers should include in their specifications and manufacturers should provide in their proposals for Thermal Storage Equipment. In addition, all of the member manufacturing companies in the ARI Thermal Storage Equipment product section are available to assist the designer in properly specifying the product.



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GUIDELINE FOR SPECIFYING THE THERMAL PERFORMANCE OF COOL STORAGE EQUIPMENT

Section 1. Purpose

1.1 *Purpose.* The purpose of this guideline is to establish the minimum information required for specifying cool storage equipment:

- a. User-specified application recommendations
- b. Supplier-specified thermal performance data

1.1.1 *Intent.* This guideline is intended for guidance of the industry, including manufacturers, engineers, installers, contractors and users.

1.1.2 *Review and Amendment.* This guideline is subject to review and amendment as technology advances.

Section 2. Scope

2.1 *Scope.* This guideline applies to Thermal Storage Equipment, for use in cooling systems, which may be charged and discharged with any of a variety of heat transfer fluids, and is either fully factory assembled, assembled on site from factory supplied components or field erected in accordance with pre-established design criteria, all as further described in Sections 3 and 4.

2.2 *Exclusions.* This guideline does not apply to Thermal Storage Equipment with thermal storage capacities of 10 ton-hours $[35 \text{ kW} \cdot h]$ or less.

Section 3. Definitions

All terms in this document will generally follow the standard industry definitions in the current edition of ASHRAE *Terminology of Heating, Ventilation, Air-Conditioning, and Refrigeration* unless otherwise defined in this section. For the convenience of the users of this guideline, definitions for the more important terms employed in this guideline are provided below:

3.1 *Ambient Air.* The air in the space surrounding the Thermal Storage Device.

3.2 *Ambient Heat Load.* The load (typically expressed in tons [kW]) imposed on the storage device due to heat gain from the Ambient Air.

3.3 *Charge Period/Cycle.* The period of time when thermal energy (heat) is removed from the storage device.

3.4 *Charge Rate.* The rate (typically expressed in tons [kW]) at which thermal energy (heat) is removed from the storage device during the charge period.

3.5 *Discharge Period/Cycle.* The period of time when thermal energy (heat) is added to the storage device.

3.6 *Discharge Rate.* The rate (typically expressed in tons [kW]) at which thermal energy (heat) is added to the storage device during the Discharge Period.

3.7 *Heat Transfer Fluid.* Any liquid used for heat transmission without a change in its phase, having no flash point or a flash point above 150°F [66°C]. Sometimes referred to as "Fluid," "Secondary Coolant" or "Coolant."

3.7.1 *Charge Fluid.* The Heat Transfer Fluid used to remove heat from a Thermal Storage Device or Thermal Storage Generator during the charge period.

3.7.2 *Discharge Fluid.* The Heat Transfer Fluid used to add heat to the Thermal Storage Device or Thermal Storage Generator during the Discharge Period.

3.8 *Ice-on-Coil.* A Thermal Storage Device consisting of coils, plates or other heat transfer surface submerged in a water-filled tank.

3.9 *Initial Charge Cycle.* The elapsed time required to bring the storage device from ambient conditions to its fully charged condition and the minimum temperature of the Heat Transfer Fluid attained during the cycle.

3.10 Latent Heat of Fusion. The change in enthalpy accompanying the conversion of a unit mass of a solid to a liquid at its melting point at constant pressure and temperature.

3.11 *Net Ice Making Capacity.* The net ice producing capability of a Thermal Storage Generator operating in a Charge mode, typically expressed in tons [kW].

3.12 *Net Storage Inventory.* General indication of the actual amount of stored cooling remaining in the storage device at a given point in the Discharge Cycle.

3.13 *Net Usable Storage Capacity.* The actual amount of stored cooling, that can be supplied from the storage device at or below the specified cooling supply temperature for a given Charge and Discharge Cycle, typically expressed in ton-hours [kW·h].

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3.14 *Nominal Storage Capacity.* A representative capacity of the storage device as defined by the storage device manufacturer (which in most cases is greater than the Net Usable Storage Capacity).

3.15 *Phase Change Material (PCM).* A substance that undergoes changes of phase while absorbing or rejecting thermal energy, normally at a constant temperature.

3.16 *Saturated Evaporator Temperature.* The dew point temperature of the refrigerant at the pressure at the outlet connection of the evaporator.

3.17 *Sensible Heat.* Heat that causes a change in fluid temperature.

3.18 *Should.* This term is used to indicate provisions which are not mandatory but which are desirable as good practice.

3.19 *Thermal Storage Device.* Equipment which stores cooling capacity using sensible and/or latent heat. May consist solely of a storage means or be packaged with one or more components of a mechanical refrigeration system.

3.20 *Thermal Storage Equipment.* Any one of, or a combination of, Thermal Storage Devices and/or Generators, that may include various other components of a mechanical refrigeration package, as indicated in Section 4. Also referred to as "Cool Storage Equipment."

3.21 *Thermal Storage Generator.* An assembly of components packaged by the manufacturer (but not necessarily shipped as one piece) typical of ice harvester or ice slurry equipment, to provide refrigeration to a Thermal Storage System. Normally includes an evaporator, and may include compressor(s), controls, heat rejection devices, etc. whose overall performance as a Thermal Storage Generator is rated by the manufacturer.

3.22 *Thermal Storage System.* All of the equipment installed to meet a specified Thermal Storage System Load (may include mechanical or absorption refrigeration equipment, see Figure 1).

3.23 *Thermal Storage System Load (or Load).* A specified cooling load to be met by the Thermal Storage System. Typically, it is expressed in tons [kW] (often referred to in this guideline as "the Load").

3.24 *Ton-Hour.* A quantity of thermal energy equal to 12,000 Btu [3.5 kW] or 1.0 ton [3.5 kW] of refrigeration provided for 1 hour.



Where:

- T₁ Temperature of coolant supplied to the Load
- T₂ Temperature of coolant returning from the Load
- T₃ Temperature of coolant entering the Cool Storage Device.
- T₄ Temperature of coolant leaving the Cool Storage Device.

Option A, B, C: Mechanical or Absorption Refrigeration Equipment (chiller)

Figure 1. Cool Storage System

Section 4. Classifications

4.1 *Classification.* Thermal Storage Equipment may be broadly classified as either "Sensible" or "Latent," with further delineations as shown in Table 1 and as explained in subsequent paragraphs.

4.2 Sensible Thermal Storage Equipment. Sensible Thermal Storage Equipment used for cooling is equipment that typically employs water as the storage medium. During the Charge Period, warm water (or other fluid) from the storage device is chilled to the desired temperature by a water chiller and returned to the storage vessel, thereby storing the energy as Sensible Heat. During the Discharge (cooling) Period, the chilled water is pumped from storage to the load and the resultant warm water returned to storage. Any of several methods may be used to keep the warm return water separated from the stored chilled water, including separate or compartmentalized tanks or where only one tank is employed, labyrinths, membranes or thermal stratification.

Table 1. Classifications of Cool Storage Equipment									
Classification	Туре	Storage Media	Charge Fluid	Discharge Fluid					
Sensible	Chilled Water or Other Fluid	Water or other fluid	Water or other fluid	Water or other fluid					
	Ice-on-Coil (External Melt)	Ice	Secondary Coolant Refrigerant	Water					
	Ice-on-Coil	Ice or other Phase Change	Secondary Coolant	Secondary Coolant					
	(Internal Meit)	Material	Refrigerant	Refrigerant					
Latent	Encapsulated Ice or Phase	Ice or other Phase Change	Secondary Coolant	Secondary Coolant					
	Change Material	Material	Water	Water					
	Ice Harvester/ Chiller	Ice	Refrigerant	Water					
	Ice Slurry	Ice in Secondary Coolant	Secondary Coolant	Secondary Coolant					

4.3 *Latent Thermal Storage Equipment.* Latent Thermal Storage Equipment may be further categorized as Ice-on-Coil, Encapsulated Ice or Phase Change Material, Ice Harvester/Chiller or Ice Slurry. Although these devices are referred to as "Latent" types, some of the stored energy is in the sensible form.

4.3.1 *Ice-on-Coil.* There are two types of Ice-on-Coil:

4.3.1.1 *External Melt.* During the Charge Period, a cold Secondary Coolant or refrigerant is circulated through the coils/plates causing ice to form on the external surfaces. Much of the liquid in the tank is not frozen. During the Discharge (cooling) Period, warm return water is circulated through the tank, external to the ice, whereby it is cooled by the melting ice.

4.3.1.2 *Internal Melt.* During the Charge Period, a cold Secondary Coolant or refrigerant is circulated through the coils/plates causing ice to form on the external surfaces. Most of the liquid in the tank is frozen. During the Discharge (cooling) Period, a warm return Secondary Coolant is circulated through the

coils/plates, internal to the ice, and cooled as the ice on the coils/plates is melted.

4.3.2 Encapsulated Ice or Phase Change Material. Thermal Storage Equipment consisting of a tank or vessel densely packed with numerous, relatively small containers in which the storage medium (water-ice or other Phase Change Material such as eutectic salt) is encapsulated. During the Charge Period, water or Secondary Coolant, at a temperature below the phase change temperature of the storage media is circulated through the tank/vessel to effect a phase change (freezing) in the storage medium. During the Discharge (cooling) Period, warm return water or Secondary Coolant is circulated through the tank/vessel and cooled as the encapsulated storage media changes phase (melts).

4.3.3 *Ice Harvester/Chiller.* Thermal Storage Equipment which employs a vertical plate, vertical tube, drum type or other evaporator in conjunction with an evaporating refrigerant to produce ice from a thin film of water flowing over the external evaporator surface. Periodically, ice is harvested by passing hot refrigerant gas through the evaporator or passing a scraper over the external surface thereby causing the ice to fall, in flakes or chunks, into a storage tank. During the Discharge (cooling) Period, warm return water is circulated through the storage

tank where it is cooled by the melting ice. Some Ice Harvesters may also serve as water chillers during the cooling period by circulating the warm return water over the evaporator where it is cooled (with or without making ice) before entering the storage tank.

4.3.4 *Ice Slurry*. Thermal Storage Equipment wherein an ice slurry is produced and stored by circulating a solution from a storage tank through an ice slurry generator. In the slurry generator, an evaporating refrigerant cools the solution, producing discrete ice crystals within the solution, which are returned to and retained within the storage tank. During the Discharge (cooling) Period, warm return Secondary Coolant is circulated through the storage tank (directly or via the Ice Slurry generator which may at that time be operating as either an ice maker or a chiller) where it is cooled by the melting ice crystals.

Section 5. Minimum Information Recommendations

5.1 User-Specified Application Recommendations. When specifying the application requirements for cool storage equipment, the user should provide, as a minimum, the following data for a design day (or design week, or other design period):

- a. Thermal Storage System Load, tons [kW], for each hour of the design day (referred to as "the Load" throughout this guideline)
- b. Usage of the Thermal Storage Equipment being used during this hour (charge, partial cooling or off) for each hour of the design day
- c. Design Heat Sink Rejection Temperature, °F [°C], (information for each hour of the design day is preferred, but not required)
- d. Supply temperature to the Load, T₁, °F[°C], during the hour of maximum load (information for each hour of the day is preferred, but not required) (Figure 1)
- e. Return temperature from the Load, T₂, °F[°C], during the hour of maximum load (information for each hour of the day is preferred, but not required) (Figure 1)
- f. Flow rate to the Load, gpm [L/s], during the hour of maximum load
- g. Maximum time, h [s], and minimum temperature, °F [°C], available to charge from fully discharged condition

 h. Identify the fluid that is flowing to the Load and the fluid flowing to the storage tanks (e.g., water, 25% ethylene glycol/75% water, etc.)

A sample format and example of the user-specified data are provided as Appendices C and D, respectively.

5.2 Supplier-Specified Thermal Performance Data. When specifying the thermal performance of cool storage equipment, the supplier should provide, as a minimum, the following data on an hourly basis for a design day (or design week, or other design period):

- a. Thermal Storage System Load, tons [kW]
- b. Load on Refrigeration Equipment, tons [kW]
- c. Thermal Storage Device Charge or Discharge Rate, tons [kW]
- Parasitic and accessory heat load (e.g., air compressor, dedicated recirculation pump, etc.) in tons [kW] into the storage device
- e. Ambient Heat Load into the storage device in tons [kW] based on defined values for:
 - 1. Ambient Air dry-bulb temperature, °F[°C]
 - 2. Ambient solar conditions (e.g., shaded, full sun, etc.)
- f. Net Storage Inventory, in the storage device, tonhour [kW·h]
- g. Saturated suction temperature and refrigeration load or other design parameters for the refrigeration plant, when this equipment is to be supplied by other than the thermal storage supplier
- h. Temperature of supply and return Fluid to the Load, T_1 and T_2 , ${}^\circ F[{}^\circ C]$
- i. Flow rate of Fluid to the Load, gpm [L/s]
- j. Temperatures of Fluid entering and leaving the Thermal Storage Device, T_3 and T_4 (Figure 1), and any other heat exchanger(s) included in the vender's scope of supply, °F [°C]
- k. Flow rate of Fluid through the Thermal Storage Device and any heat exchanger(s) included in the supplier's scope of supply, gpm [L/s]
- 1. Pressure drop across the Thermal Storage Device and any heat exchanger(s) included in the supplier's scope of supply, psi [kPa]

- m. Energy input to thermal storage refrigeration equipment included in the supplier's scope of supply, kWh [kWh] (for electric chiller) or kBtu [kWh] (for gas-fired chiller)
- Total heat rejection, Btu [kW] and condensing n. temperature for the refrigeration system if within the supplier's scope of supply. If the heat rejection device is included in the supplier's scope of supply, the temperature, ${}^{\circ}F[{}^{\circ}C]$ (and flow rate, gpm [L/s], if applicable) of the heat rejection sink, °F [°C] - e.g. condenser water supply temperature and flow rate for water cooled condensers, entering dry-bulb temperature for air-cooled condensers, entering wet-bulb temperature evaporative for condensers, etc.
- o. Energy input to essential storage device parasitics and accessories, i.e. air compressors or air pumps, kWh

Note: It is intended that hour-by-hour changes in Net Storage Inventory track the hour-by-hour effects of Charge Rate, Discharge Rate, parasitic load and Ambient Heat Load. As such, the data should represent a theoretically endlessly repeatable cycle for back-to-back design days, weeks, or whatever basis is chosen for the cycle. Additional minimum data to be supplied by the supplier includes:

- a. Listing of all equipment included in the scope of supply
- b. Net Usable Storage Capacity, ton-hour [kW·h] for the Thermal Storage Device(s)
- c. Time, ,h [s], required to charge from the fully discharged state
- d. Time, h [s], required to recharge after discharge on design day (for purposes of this guideline, the design day is to be considered to be the peak day unless otherwise specified by the design engineer)
- e. Identification of the Charge and Discharge Fluids, including the physical properties if these are not readily available from conventional sources

Also, note that for ice slurry systems it may be necessary to define the Secondary Coolant concentration at a defined point of time within the operating cycle.

A sample format and example of the supplier-specified data are provided as Appendices E and F respectively.

APPENDIX A. REFERENCES - NORMATIVE

None.

APPENDIX B. REFERENCES - INFORMATIVE

B1 Listed here are standards, handbooks, and other publications which may provide useful information and background but are not considered essential.

B1.1 ARI Standard 900-98, *Thermal Storage Equipment used for Cooling*, 1998, Air-Conditioning and Refrigeration Institute, 4100 N. Fairfax Drive, Ste. 200, Arlington, VA 22203, U.S.A.

B1.2 ASHRAE Design Guide for Cool Thermal Storage, 1993, American Society of Heating, Refrigerating, and Air-Conditioning Engineers, Inc., 1791 Tullie Circle, N.E., Atlanta, GA 30329, U.S.A.

B1.3 ASHRAE Handbook – HVAC Applications, Chapter 33, "Thermal Storage", 1999, American Society of Heating, Refrigerating, and Air-Conditioning Engineers, Inc., 1791 Tullie Circle, N.E., Atlanta, GA 30329, U.S.A. **B1.4** ASHRAE Successful Cool Storage Projects: From Planning to Operation, 1996, American Society of Heating, Refrigerating, and Air-Conditioning Engineers, Inc., 1791 Tullie Circle, N.E., Atlanta, GA 30329, U.S.A.

B1.5 ASHRAE Standard 150-2001, *Method of Testing the Performance of Cool Storage Systems*, 2001, American Society of Heating, Refrigerating, and Air-Conditioning Engineers, Inc., 1791 Tullie Circle, N.E., Atlanta, GA 30329, U.S.A.

B1.6 ASHRAE Terminology of Heating, Ventilation, Air Conditioning, & Refrigeration, Second Edition, 1991, American Society of Heating, Refrigerating, and Air-Conditioning Engineers, Inc., 1791 Tullie Circle, N.E., Atlanta, GA 30329, U.S.A.

APPENDIX C. USER-SPECIFIED APPLICATION RECOMMENDATIONS DATA - INFORMATIVE

Table C1. Recommended Specification Information - Example						
Fluid used to define the following design data						
Supply Temperature to Load at peak conditions, T_1 , ${}^{\circ}F[{}^{\circ}C]$						
Return Temperature from Load at peak conditions, T ₂ , °F [°C]						
Flow rate to Load at peak conditions, gpm [L/s]						
Maximum allowable pressure drop through storage device, psi [kPa]						
System Schematic (attached to data sheets)						
Secondary Coolant (if applicable)						
Maximum time and minimum temperature available to charge Thermal						
Storage Device from fully discharged condition						
(Initial Charge Cycle), h [s] and °F [°C]						
Design Heat Sink Rejection Temperature, °F [°C]						

[NOTE: Shaded areas in Tables C1 and C2 are optional]

	Table C2. Preferred User-specified Data - Example Design Day										
Hour	Thermal Storage System Load tons [kW]	Supply Temperature to Load, T ₁ °F [°C]	Return Temperature from Load, T ₂ °F [°C]	Flow Rate to Storage System gpm [L/s]	Heat Sink Rejection Temperatures (Wet-Bulb or Dry-Bulb) °F [°C]	Thermal Storage Refrigeration Equipment Use during this hour? (Charge / Partial Cooling / Off)					
0 - 1											
1 - 2											
2 - 3											
3 - 4											
4 - 5											
5 - 6		1									
6 - 7											
7 - 8											
8 - 9											
9 - 10											
10 - 11											
11 - 12											
12 - 13											
13 - 14											
14 - 15											
15 - 16											
16-17											
17 - 18											
18 - 19											
19 - 20											
20 - 21											
21 - 22											
22 - 23											
23 - 0											
Totals											

APPENDIX D. SAMPLE USER-SPECIFIED APPLICATION RECOMMENDATIONS DATA – INFORMATIVE

Table D1. Recommended Specification Information - Sample						
Fluid Used to define the following design data	25% Ethylene Gl	ycol/75% Water				
Supply Temperature to Load at peak conditions, T ₁ , °F [°C]	44°F					
Return Temperature from Load at peak conditions, T ₂ , °F [°C]	58°F					
Flow rate to Load at peak conditions, gpm [L/s]	1822 gpm					
Maximum allowable pressure drop through storage device, psi [kPa]	14 psi					
System Schematic (attached to data sheets)	Yes					
Secondary Coolant (if applicable)	Same as Above					
Maximum time and minimum temperature available to charge Thermal	16 hours	22 °F				
Storage Device from fully discharged condition (Initial Charge Cycle), h						
[s] and °F [°C]						
Design Heat Sink Rejection Temperature °F [°C]						

[NOTE: Shaded areas in Tables D1 and D2 are optional]

	Table D2. Preferred User-specified Data - Sample Design Day									
Hour	Thermal Storage System Load tons	Supply Temperature to Load, T ₁ °F	Return Temperature from Load, T ₂ °F	Flow Rate to Storage System gpm	Heat Rejection Temperatures (Wet-Bulb or Dry-Bulb) °F	Thermal Storage Refrigeration Equipment Use during this hour? (Charge/Partial-Cooling/ Off)				
0 - 1	0					Charge				
1 - 2	0					Charge				
2 - 3	0					Charge				
3 - 4	0					Charge				
4 - 5	0					Charge				
5 - 6	0					Charge				
6 - 7	0					Charge				
7 - 8	800	44.0	55.2	1822		Partial Cooling				
8 - 9	700	44.0	53.8	1822		Partial Cooling				
9 - 10	600	44.0	52.4	1822		Partial Cooling				
10 - 11	700	44.0	53.8	1822		Partial Cooling				
11 - 12	800	44.0	55.2	1822		Partial Cooling				
12 - 13	900	44.0	56.6	1822		Off				
13 - 14	1000	44.0	58	1822		Off				
14 - 15	1000	44.0	58	1822		Off				
15 - 16	900	44.0	56.6	1822		Off				
16-17	800	44.0	55.2	1822		Partial Cooling				
17 - 18	700	44.0	53.8	1822		Partial Cooling				
18 - 19	0					Charge				
19 - 20	0					Charge				
20 - 21	0]				Charge				
21 - 22	0					Charge				
22 - 23	0					Charge				
23 - 0	0					Charge				
Totals	8900 ton-h									

APPENDIX E. SUPPLIER-SPECIFIED THERMAL PERFORMANCE DATA - INFORMATIVE

					Example Design I	Day					
		Net Usa	ble Storage Ca	pacity:		Ton-Hours [k]	W∙ h] (Total C	olumn D)			
Heat Tra Specific (Specific H	nsfer Fluid: Gravity: Heat Btu/lb/°F [kJ	@ J/kg·K]:	°F [°C] @	°F [°C]		Hours to Recharge fr Hours to Recharge or	om Fully Dischar; n Design Day:	ged Condition: _ hours	hours		
	Table E1. Supplier-specified Data										
Hour	<u>A</u> Thermal Storage System Load tons [kW]	<u>B</u> Refrigeration Equipment Load tons [kW]	<u>C</u> Storage Device Charge Rate tons [kW]	<u>D</u> Storage Device Discharge Rate* tons [kW]	<u>E</u> Parasitic and Accessory Heat Load into Storage Device tons [kW]	<u>F</u> Ambient Heat Load into Storage Device tons [kW]	<u>G</u> Net Storage Inventory** Ton-Hours [kW ⋅h]	$\begin{array}{c} \underline{H} \\ Supply \\ Temperature \\ to Load, T_1 \\ ^{\circ}F [^{\circ}C] \end{array}$	<u>I</u> Return Temperature from Load, T ₂ °F [°C]	<u>J</u> Flow Rate to Load gpm [L/s]	
0 - 1											
1 - 2											
2 - 3											
4-5											
5-6											
6 - 7											
7 - 8											
8 - 9											
9 - 10											
10 - 11											
11 - 12											
12 - 13											
13 - 14											
14 - 15											
15 - 10											
17 - 18											
18 - 19											
19 - 20											
20 - 21											
21 - 22											
22 - 23											
23 - 0											
Totals									-		

* Greater Discharge Rates may not be possible at defined discharge temperature (T_4)

** Net Storage Inventory values are not available for instantaneous discharge

For Design to work, all of the following must be true:

1. Totals for column B must be greater than or equal to the sum of totals for columns A, E and F.

2. The values in Column I must always be less than maximum temperature defined on the "User-Specified Data" Sheet.

	K	L	M	N	<u>0</u>	Р	Q	R	S
	Fluid Temp.	Fluid Temp.	Flow Rate	Pressure Drop	Storage Device	Saturated Suction	Condenser:	Refrigeration	Storage
	Entering	Leaving Storage	Through	for Storage	Refrigeration Energy	Temp. ***	Heat Load,	Condensing	Device
Hour	Storage Device	Device	Storage Device	Device	Input, kWh (electric	°F [°C]	Tons [kW] or	Temp. °F [°C] or	Parasitics
	T ₃	T_4	gpm [L/s]	psi [kPa]	chiller) or kBtu (gas-		Water Flow	Condenser Heat	Electrical
	°F [°C]	°F [°C]			fired chiller)		gpm [L/s]****	Sink Temp	Input (kWh)
								°F [°C]****	
0 - 1									
1 - 2									
2 - 3									
3 - 4									
4 - 5									
5 - 6									
6 - 7									
7 - 8									
8 - 9									
9 - 10									
10 - 11									
11 - 12									
12 - 13									
13 - 14									
14 - 15									
15 - 16									
16 - 17									
17 - 18									
18 - 19									
19 - 20									
20 - 21									
21 - 22									
22 - 23									
23 - 0									
Totals									

 k*** Ice Harvester and Slurry systems only.
 Applicable where refrigeration equipment is within the thermal storage supplier-s scope.
 k**** Specify Heat Sink Type and Temperature: (Condenser Water Temp.; Dry Bulb Temp.; Wet Bulb Temp.)

APPENDIX F. SAMPLE SUPPLIER-SPECIFIED THERMAL PERFORMANCE DATA - INFORMATIVE

Example Design Day

 Net Usable Storage Capacity:
 4700
 Ton-Hours (Total Column D)

 Heat Transfer Fluid:
 25% EG / H₂O
 F
 Hours to Recharge from Fully Discharged Condition:
 15

 Specific Gravity:
 1.027
 @
 60
 °F
 Hours to Recharge from Fully Discharged Condition:
 15

 Specific Heat Btu/lb/°F:
 0.93
 @
 60
 °F
 Hours to Recharge on Design Day:
 13
 hours

Table F1. Supplier-specified Data - Sample B С D E F G Η Α Ι J Storage Device Parasitic and Net Storage Thermal Refrigeration Storage Device Ambient Heat Supply Return Flow Rate Storage System Equipment Charge Rate Discharge Rate* Accessory Heat Load Load into Storage Inventory** Temperature Temperature to Load Hour Load Load tons tons into Storage Device Device Ton-Hours to Load, T₁ from Load, T₂ gpm tons tons tons tons °F °F 0 - 1 0 390 2 2730 31.4 31.4 0 390 1 - 2 390 2 3120 0 390 31.2 31.2 0 2 - 3 0 390 2 3510 30.9 30.9 0 390 3 - 4 0 390 2 3900 30.4 30.4 0 390 4 - 5 0 390 2 4290 29.7 29.7 0 390 5 - 6 2 4680 28.9 28.9 0 390 390 0 6 - 7 0 27.8 390 2 5070 27.8 0 390 7 - 8 800 600 200 2 4870 44.0 55.2 1821 8-9 700 600 100 2 4770 44.0 53.8 1821 9 - 10 600 4770 52.4 1821 600 0 2 44.0 10 - 11 700 600 100 2 4670 44.0 53.8 1821 11 - 12 55.2 800 600 200 2 4470 44.0 1821 12 - 13 900 0 900 2 3570 44.0 56.6 1821 13 - 14 1000 0 1000 2 2570 44.0 58.0 1821 14 - 15 1000 0 1000 2 1570 44.0 58.0 1821 15 - 16 900 0 900 2 670 44.0 56.6 1821 16-17 800 600 200 2 470 44.0 55.2 1821 17 - 18 700 600 100 2 370 44.0 53.8 1821 18 - 19 0 390 2 390 31.8 31.8 0 390 19 - 20 0 390 2 780 31.7 31.7 0 390 20 - 21 0 390 390 2 1170 31.7 31.7 0 21 - 22 0 390 2 1560 31.6 31.6 0 390 22 - 23 0 2 0 390 1950 31.6 31.6 390 23 - 0 0 390 2 2340 31.5 31.5 0 390 Totals 8900 9270 5070 4700 48

* Greater Discharge Rates may not be possible at defined discharge temperature (T₄)

** Net Storage Inventory values are not available for instantaneous discharge

For Design to work, all of the following must be true:

1. Totals for column B must be greater than or equal to the sum of totals for columns A, E and F.

2. The values in Column I must always be less than maximum temperature defined on the "User-Specified Data" Sheet.

hours

	<u>K</u>	<u>L</u>	M	<u>N</u>	<u>0</u>	<u>P</u>	<u>Q</u>	<u>R</u>	<u>S</u>
	Fluid Temp.	Fluid Temp.	Flow Rate	Pressure Drop	Storage Device	Saturated	Condenser: Heat	Refrigeration	Storage Device
	Entering Storage	Leaving Storage	Through Storage	for Storage	Refrigeration	Suction	Load, Tons or	Condensing	Parasitics
Hour	Device	Device	Device	Device	Energy Input,	Temp. ***	water Flow	Condenser Heat	(kWb)
	13 °F	T_4	gpin	psi	chiller) or kBtu	°Р	gpin	Sink Temp	(KWII)
	-	Г			(gas-fired chiller)			°F****	
0 - 1	26.0	31.4	1821	9.3					
1 – 2	25.8	31.2	1821	9.4					
2 - 3	25.4	30.9	1821	9.4					
3 - 4	24.9	30.4	1821	9.4					
4 - 5	24.3	29.7	1821	9.4					
5 - 6	23.4	28.9	1821	9.5					
6 - 7	22.3	27.8	1821	9.5					
7 - 8	46.8	32.1	347	1.2					
8 - 9	45.4	32.1	192	0.6					
9 - 10	44.0	32.1	0	0.0					
10 - 11	45.4	32.2	192	0.6					
11 - 12	46.8	32.3	353	1.2					
12 - 13	56.6	36.2	1128	4.4					
13 - 14	58.0	38.8	1330	5.4					
14 - 15	58.0	41.2	1519	6.4					
15 - 16	56.6	43.7	1773	7.9					
16 – 17	46.8	39.6	710	2.6					
17 - 18	45.4	41.6	679	2.4					
18 - 19	26.4	31.8	1821	9.3					
19 - 20	26.3	31.7	1821	9.3					
20 - 21	26.2	31.7	1821	9.3					
21 – 22	26.2	31.7	1821	9.3					
22 - 23	26.2	31.6	1821	9.3					
23 - 0	26.1	31.6	1821	9.3					
Totals									

Ice Harvester and Slurry systems only. Applicable where refrigeration equipment is within the thermal storage suppliers scope. Specify Heat Sink Type and Temperature: (Condenser Water Temp.; Dry Bulb Temp.; Wet Bulb Temp.) ****